

Chapter 2 Contact phenomena

MSE 485 Tribology

- 1. Adhesion
- 2. Contact mechanics
- 3. Surface topography

Adhesion

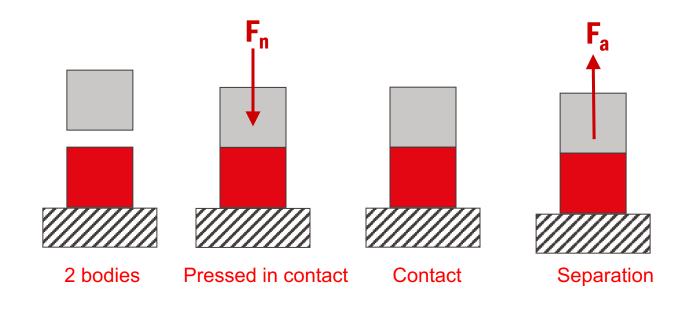
Bonding at the interface between two bodies brought into contact.

— Interatomic bond
— Lone pair (high energy)
Adhesion!

The degree of adhesion depends on:

- Type of bonding: ionic, covalent, metallic, Van der Waals (dipoles)
- Malleability of the materials
- Surface roughness, cristallographic orientation
- Contaminants (oxidation, adsorption of molecules)

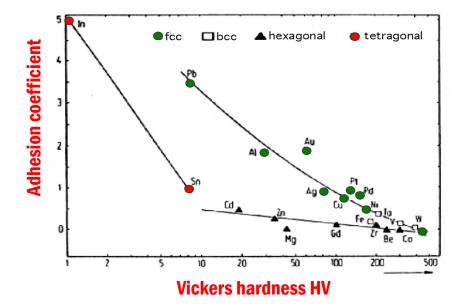
Adhesion test



Coefficient of adhesion a:
$$a = \frac{F_a}{F_n} - 1$$

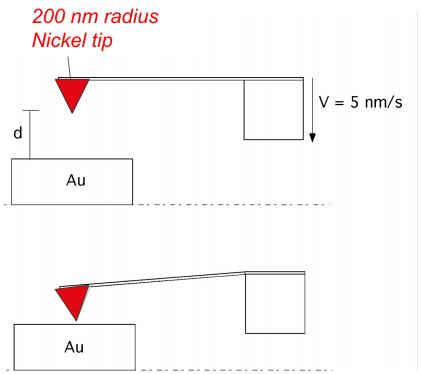
Adhesion coefficient

 Adhesion coefficient of pure metals against themselves as a function of hardness and cristalline structure :



Nanoscopic aspects

• Experience with an AFM type tool: (Landman et al, Science 248, 454-461, 1990)



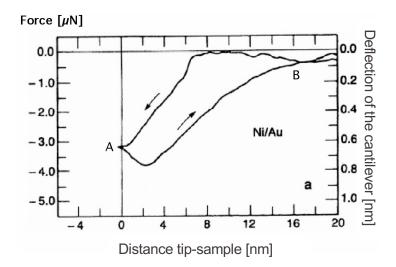
Approach of the Ni tip towards the Au sample at a constant speed.

Deflection of the cantilever due to the attractive force between the Ni tip and the Au sample.

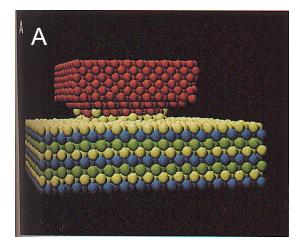
EPFL

Case of a contact without indentation

• Experimental curves: force versus distance:



- Numerical simulation :
- (molecular dynamics based on Lennard-Johnes' potential)



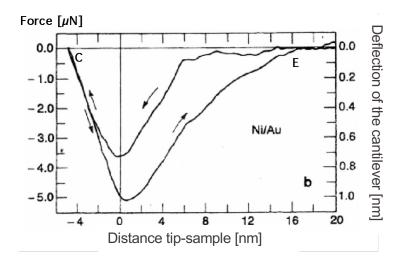
Red: Ni atoms

Yellow/Blue/Green: Au atoms

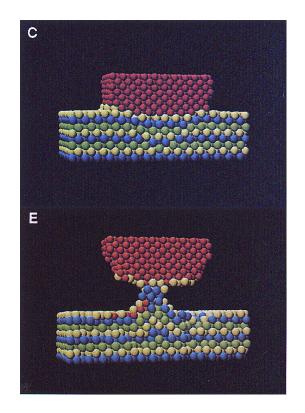


Case of a contact with indentation

Experimental curves: force versus distance:



Numerical simulation :



1. Adhesion

2. Contact mechanics

3. Surface topography

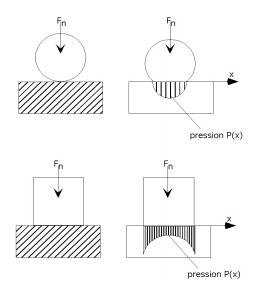


Study of the amplitude and distribution of mechanical stresses in a contact.

Conformity of the contact

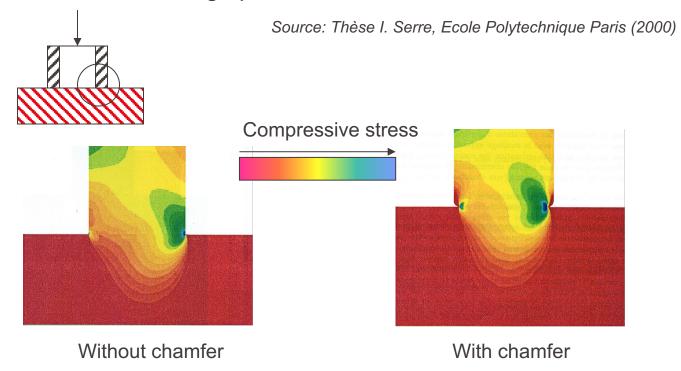
Non-conformal contact:

Conformal contact:



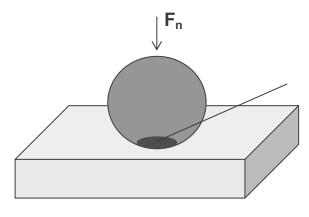
Conformal contact

 Example of finite elements calculation of stress distribution in a conformal contact : rings-plane contact



Analysis of elastic stress fields. Non-conformal contacts

- Hertz mechanics for non-conformal contacts:
 - Calculation of elastic strain and stress in terms of load, geometrical parameters and materials.



Elastic deformation defines the contact area.

Ball-plane contact

 $\frac{1}{E'} = 0.5 \left| \frac{1 - v_1^2}{E_1} + \frac{1 - v_2^2}{E_2} \right|$

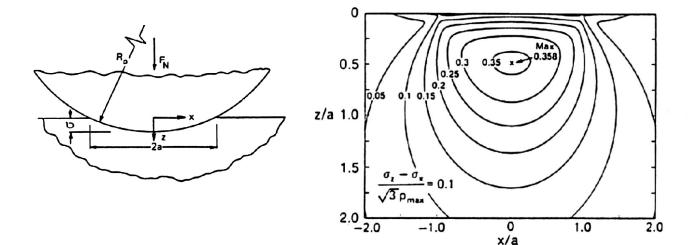
E = Young's modulusv = Poisson's ratio

- Radius of contact area (circle) $a = \left(\frac{1.5F_nR}{E'}\right)^{\frac{1}{3}}$
- Maximum contact pressure $p_0 = \frac{3F_n}{2\pi a^2}$
- Average contact pressure $p_m = \frac{F_n}{\pi a^2}$
- Maximum deflection $w = 1.31 \left(\frac{F_n^2}{E^{\prime 2} R} \right)^{\gamma_3}$
- Maximum shear stress $\tau_{\text{max}} = \frac{p_0}{2}$
- Depth of maximum shear strength

$$z = 0.638 \cdot a$$

Ball-plane contact

Distribution of the shear stresses in a ball-plane contact :



The maximum shear stress is located below the surface and corresponds to 1/3 of maximum pressure.

Onset of plastic deformation

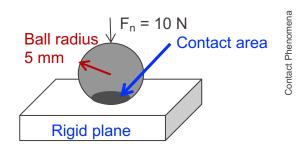
• Tresca criterion: plastic deformation occurs when a critical shear stress of YS /2 (YS: Yield strength in uniaxial traction) is reached.

For a ball on plane contact plastic deformation occurs when:

$$P_0 > 1.61 \, YS$$

$$P_m > 1.07 \text{ YS}$$

Application to different materials

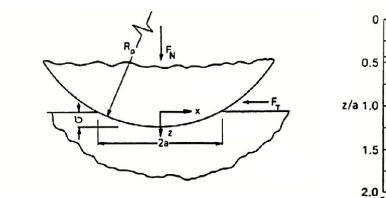


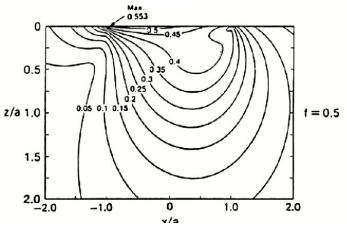
Ball Material	Elastomer	Polymer	Metal	Ceramic	Unit
E Modulus	0.02	1	200	500	GPa
Poisson ratio	0.5	0.5	0.3	0.3	
Radius of contact area	1.121	0.304	0.057	0.043	mm
Average Pressure	3	34	995	1740	MPa
Yield Strength (σ_{Y})	10	20	350	350	MPa
Av. Pressure/ $\sigma_{\rm Y}$	0.3	1.7	2.8	5.0	

Softer but more elastic materials can accommodate contact stresses without plastic deformation

Effect of tangential force

Ball-plane contact : numerical calculation

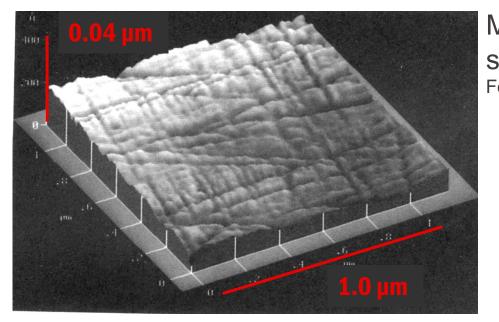




An additional tangential load (friction) yields larger shear stresses and shifts the maximum shear stress towards the surface

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Surface roughness

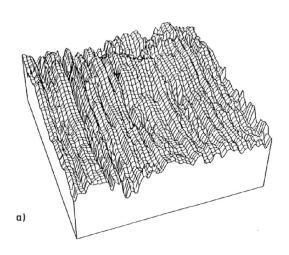


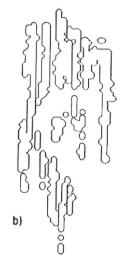
"Putting two solids together is rather like turning Switzerland upside down and standing it on Austria – the area of intimate contact will be small"

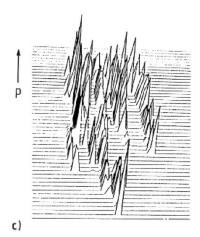
F.P. Bowden

Effect of roughness on local contact pressure

- Numerical simulation of a model Hertzian contact with a rough steel surface ($F_n = 25 \text{ N}$, $p_0 = 1 \text{ GPa}$, elliptical contact area : semi-axes 78 µm and 162 µm).
 - a) Representation of the steel surface : area 0.5 mm2, maximum relief 4.4 μm)
 - b) Contour of the contact area
 - c) Pressure distribution (maximum value **7 GPa**)







Plasticity index Y

Contact between material 1 (rough, soft) and material 2 (smooth, rigid):

(Greenwood & Williamson, 1966) $Y = E'H^{-1} \left(\frac{\sigma}{R}\right)^{0.5}$

Where:

- H = hardness of material 1
- σ = standard deviation of the distribution of the heights of the roughness profile
- R = mean radius of asperities
- $(\sigma / R)^{0.5}$ = mean slope of asperities (< 0.2 for technical surfaces)
- E' is defined as : $\frac{1}{E'} = 0.5 \left[\frac{1 v_1^2}{E_1} + \frac{1 v_2^2}{E_2} \right]$

Y < 0.6 : elastic deformation of asperities

Y > 1 : plastic deformation of asperities

Application to technical metallic surfaces

• Simplifications : E1 = E2

$$v^2 \approx 0$$

$$(\sigma / R)^{0.5} = 0.2$$

 $Y \approx 0.1 E/H$

Approximation for metals :

 E range from 70 GPa to

> 10 GPa from 1 GPa to

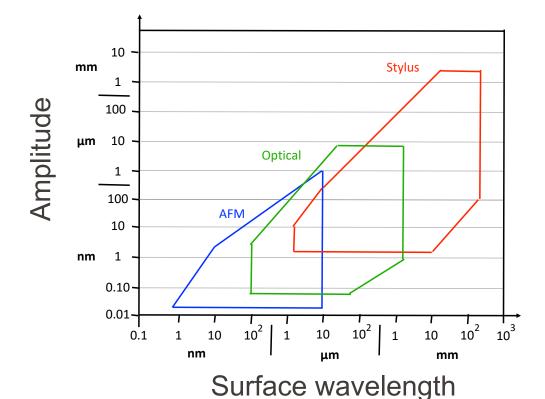
H range

Y > 0.7

For typical values of surface finish after machining, the deformation of asperities is almost always plastic!

200 Gpa

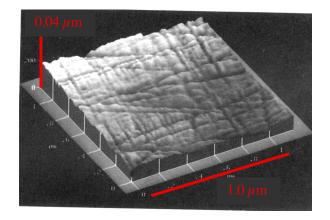
Surface topography: measurement techniques



Profilometry vs Atomic force microscopy

Resolution

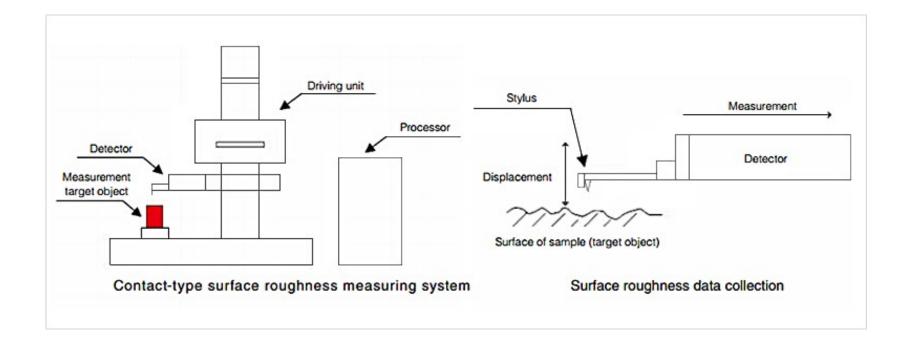
Technique Direction	Profilometer	AFM			
Horizontal	1000 nm	10 nm			
Vertical	10 nm	0.1 nm			



Mirror polished steel surface: AFM image

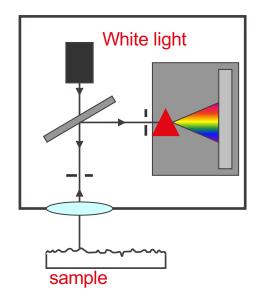


Measurement techniques: stylus profilometer

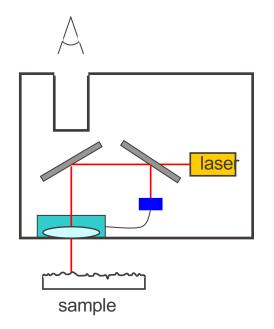


Measurement techniques: non-contact profilometer

Chromatic confocal

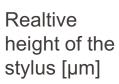


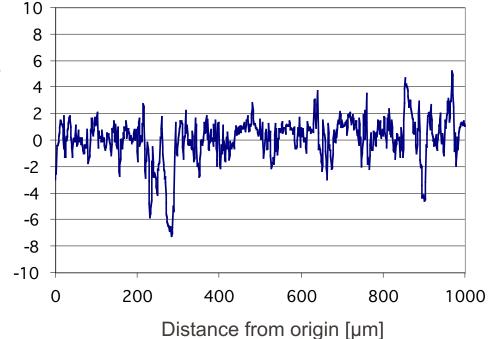
Dynamic focusing



Measurement example

Two-dimensional topographic measurement:

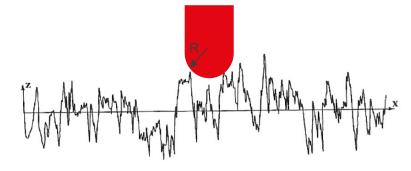




Size of the probe

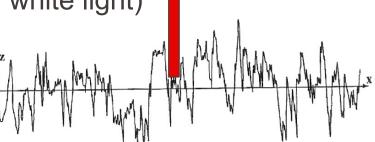
Effect of the probe size R on profile measurement :
 Stylus:

 $R = 1-2.5 \mu m$



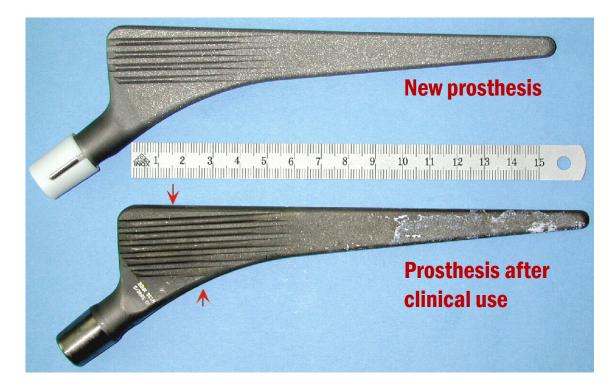
Optical beam : (laser or white light)

 $R = 0.5-1 \mu m$



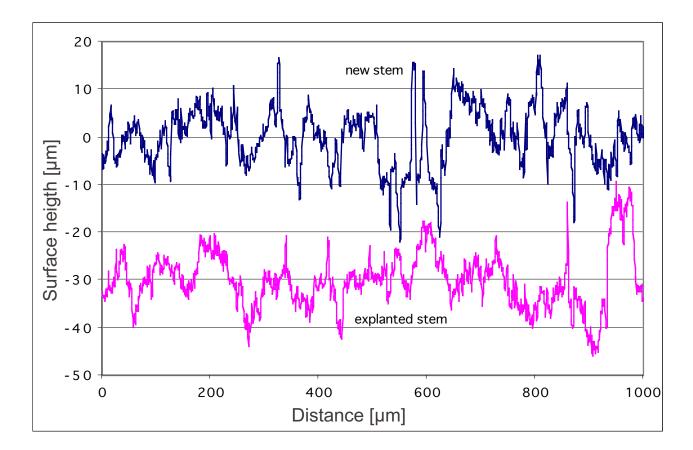
Hip implant stem

 Comparison : new stem vesus stem after 15 years of clinical use (titanium)



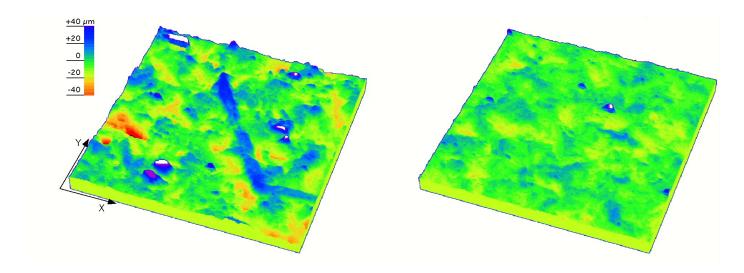


Two-dimensional profilometry



Tri-dimensional profilometry

• Analysed surface : 0.5 x 0.5 mm²:



New prosthesis

Prosthesis after clinical use

Surface topography: characterization of real surfaces

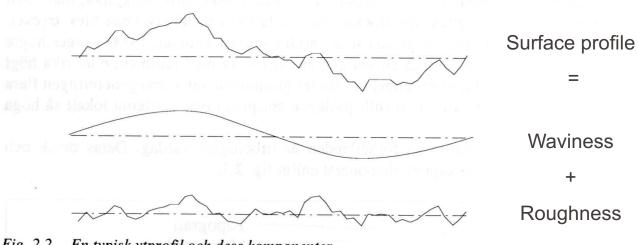


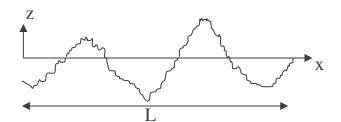
Fig. 2.2 En typisk ytprofil och dess komponenter.

Waviness can be caused by the machining process – a systematic error (vibrations/deflections).

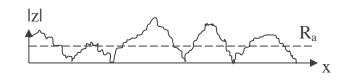
The roughness is a random error produced when material is removed from the surface.

Definition of the most usual parameters : R_a et R_q

• R_a: central line average roughness



 $R_a = \frac{1}{n} \sum_{i=1}^n \left| z_i \right|$



• R_q: root mean square roughness

$$R_{q} = \sqrt{\frac{1}{n}} \sum_{i=1}^{n} Z_{i}^{2}$$

z : height of the profile at position x

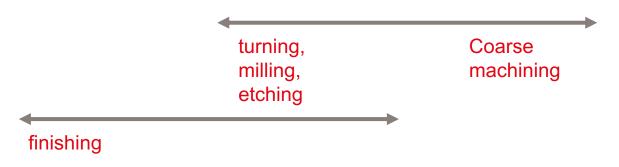
n: number of measurement points over the length L

L: measurement length

Technological aspects

• The *roughness* of technical pieces is linked to the *machining quality* grade :

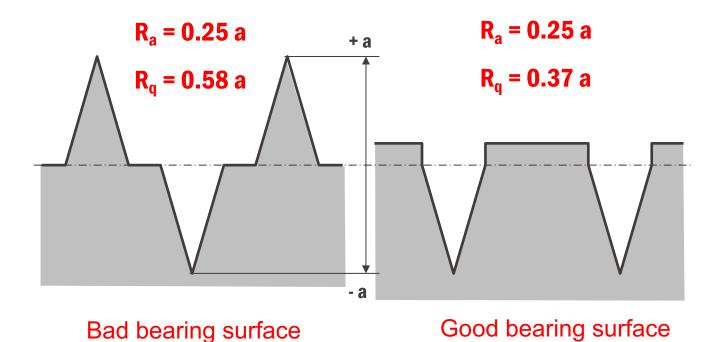
Grade	N1	N2	N3	N4	N5	N6	N7	N8	N9	N10	N11	N12
$R_a \mu m$	0.025	0.05	0.1	0.2	0.4	0.8	1.6	3.2	6.3	12.5	25	50



for R_a<0.025 µm : polishing

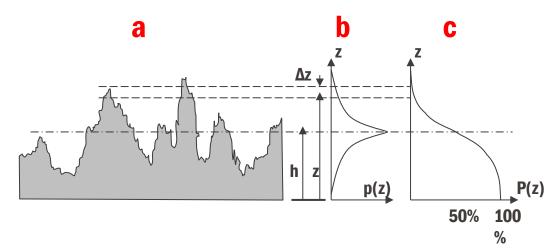
Limitations of height parameters

Different surface qualities yield the same R_a value :



Statistical parameters

- a) Surface profile
- b) Height density of probability function (probability of finding a point of the surface in within a Δz interval and at a height z from the central line)
- c) Cumulative probability function (probability of finding a point of at a height higher than z)



Statistical parameters

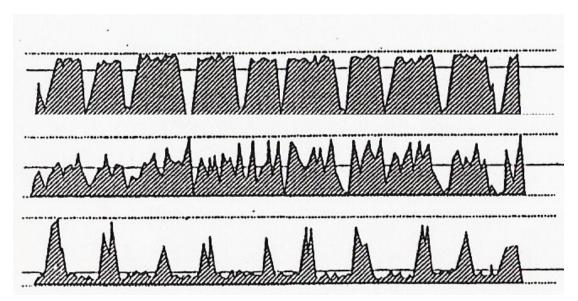
Sk :Skewness (asymmetry) of the distribution profile p(z)

$$S_{k} = \frac{1}{n} \sum_{i=1}^{n} \frac{Z_{i}^{3}}{R_{q}^{3}}$$



$$S_k = 0$$

$$S_k > 0$$



Statistical parameters

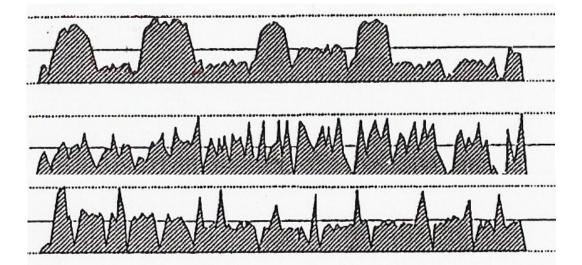
K: Kurtosis (sharpness) of the distribution profile p(z)

$$K = \frac{1}{n} \sum_{i=1}^{n} \frac{Z_{i}^{4}}{R_{q}^{4}}$$









Case of the hip prosthesis stem



Mean and standard deviation over five profiles :

	Implant neuf	Implant après utilisation
$\mathbf{R}_{\mathbf{a}}$	4.8 ± 0.6	2.7 ± 0.2
$\mathbf{R}_{\mathbf{q}}$	6.1 ± 0.7	3.5 ± 0.2
R_z	26 ± 2	16 ± 1
S_k	$0.1\ \pm\ 0.4$	0 ± 0.2
K	3.2 ± 0.6	3.5 ± 0.6